

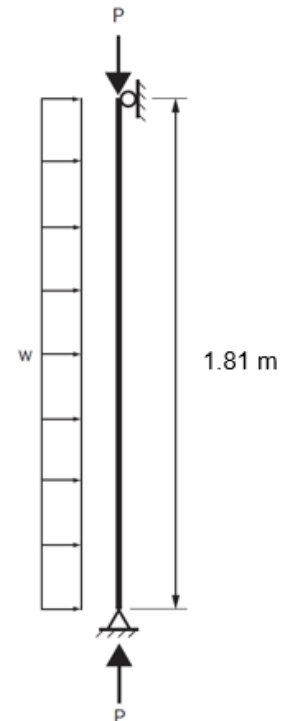
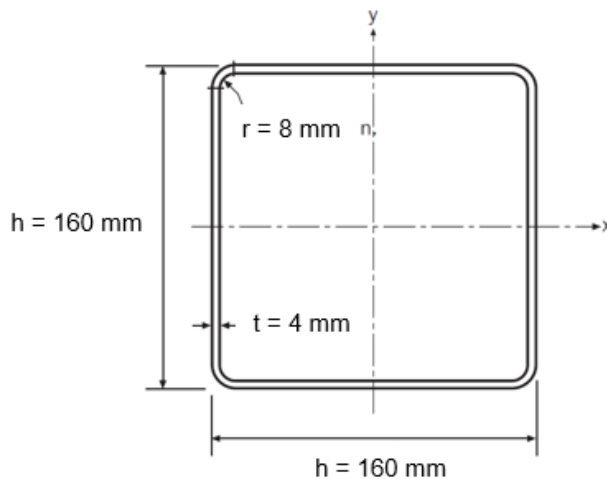
EC3 1-3 2006 CFFD Example 003

BOX-SECTION MEMBER UNDER COMBINED COMPRESSION, BENDING, AND SHEAR

EXAMPLE DESCRIPTION

Compression, moment, and shear capacities and demand/capacity ratio are calculated for box section at mid-height as shown below. It is simply supported with a length of 1.81 meters.

GEOMETRY, PROPERTIES AND LOADING



Dead: $P = 400,000 \text{ N}$, $w = 20 \text{ N/mm}$

TECHNICAL FEATURES TESTED

- Axial compressive strength
- Major moment strength
- Shear strength
- Demand/Capacity ratio.

COMPUTER FILE: EC3 1-3 2006 CFFD Ex003

Applicable Programs

➤ SAP2000

RESULTS COMPARISON

Independent results are hand calculated.

CONCLUSION

The results show exact match with independent results.

Benchmarks: SAP2000

Output Parameter	Program	Independent	Percent Difference
Axial - Flexural buckling $N_{b,Rd} (N)$	731486	731477	0.00%
Axial – Local & Distortional Buckling $N_{c,Rd} (N)$	776251	776243	0.00%
Flexure – Lateral-Torsional Buckling $M_{b,Rd} (N - mm)$	42501809	42501807	0.00%
Flexure – Local & Distortional Buckling $M_{c,Rd} (N - mm)$	42501809	42501807	0.00%
Shear $V_{b,Rd} (N)$	256963	256963	0.00%
D/C Ratio	0.885	0.885	0.00%

HAND CALCULATION

Properties:

Material: $E = 210,000 \text{ N/mm}^2$, $G = 80,770 \text{ N/mm}^2$, $f_{yb} = 355 \text{ N/mm}^2$

Section:

$$h = b = 160 \text{ mm}, t = 4 \text{ mm}, r = 8 \text{ mm}$$

$$\rightarrow h_p = b_p = h - t = 160 - 4 = 156 \text{ mm}$$

Check for the effect of rounding of the corners:

$$\frac{r}{t} = \frac{8}{4} = 2 < 5 \rightarrow OK$$

$$\frac{r}{b_p} = \frac{8}{156} = 0.051 < 0.1 \rightarrow OK$$

Therefore, the effect of rounding of the corners can be neglected in calculation of section properties:

$$A_g = 2496 \text{ (mm}^2\text{)}$$

$$I_y = I_z = 10122763.6 \text{ (mm}^4\text{)}$$

$$i_y = i_z = 63.684 \text{ (mm)}$$

$$W_{el} = 129779 \text{ (mm}^3\text{)}$$

$$I_t = 15185664 \text{ (mm}^4\text{)}$$

$$y_0 = z_0 = 0.0 \text{ (mm)}$$

Member: $K_y = K_z = 0.9$ to model sway condition according to Annex BB.1.3(1) of EN1993-1-1

$$L_y = L_z = 1810 \text{ mm}$$

$$k_{yy} = k_{zz} = k_{zy} = k_{yz} = 1.0$$

Loadings: Dead: $P = 400,000 \text{ N}$, $w = 20 \text{ N/mm}$

Required strengths: for the section in the middle

$$N_{Ed} = P = 400,000 \text{ (N)}$$

$$M_{Ed} = \frac{20 \times 1810^2}{8} = 8190250 \text{ (N} \cdot \text{mm)}$$

$$V_{Ed} = 0.0 \text{ (N)}$$

Member Compression Capacity: the compression capacity is calculated considering the limit states of global buckling, and local buckling. Distortional buckling is not considered because it is closed-box section.

1. Local buckling:

The effective width method is utilized to calculate the nominal axial strength in consideration of local buckling with the compressive stress of $f_{yb} = 355 \text{ (N/mm}^2\text{)}$.

Check for the applicability of the method as the following conditions are satisfied:

$$\frac{h}{t} = \frac{b}{t} = \frac{160}{4} = 40 < 60 \rightarrow OK$$

As the section is subjected to uniform compression and all four (4) sides are identical, they are considered internal stiffened elements:

$$\psi = 1$$

$$k_\sigma = 4$$

$$\varepsilon = \sqrt{\frac{235}{f_{yb} [N/mm^2]}} = \sqrt{\frac{235}{355}} = 0.8136$$

$$\bar{\lambda}_{p,b} = \frac{b_p/t}{28.4\varepsilon\sqrt{k_\sigma}} = \frac{156/4}{28.4 \times 0.8136\sqrt{4}} = 0.844 > 0.673$$

$$\rho = \frac{\bar{\lambda}_{p,b} - 0.055(3 + \psi)}{\bar{\lambda}_{p,b}^2} = \frac{0.844 - 0.055(3 + 1)}{0.844^2} = 0.876 \leq 1.0$$

$$b_{eff} = \rho b_p = 0.876 \times 156 = 136.66 \text{ (mm)}$$

$$A_{eff} = 4tb_{eff} = 4 \times 4 \times 136.66 = 2186.6 \text{ (mm}^2\text{)}$$

$$A_{eff} = 2186.6 \text{ (mm}^2\text{)} < 2496 \text{ (mm}^2\text{)} = A_g$$

$$\rightarrow N_{c,Rd} = \frac{A_{eff}f_{yb}}{\gamma_{M0}} = \frac{2186.6 \times 355}{1.0} = 776243 \text{ (N)}$$

Because the section is doubly-symmetric, its effective properties are also symmetric about y-y and z-z axis, resulting in $e_{Ny} = e_{Nz} = 0 \rightarrow \Delta M_{y,Ed} = \Delta M_{z,Ed} = 0$

2. Global buckling: includes flexural buckling. Torsional and flexural-torsional buckling is not considered because it is box section

- i. Flexural buckling: since the member is doubly symmetric, flexural buckling strength is identical for y-y and z-z axis.

$$N_{cr,y} = \frac{\pi^2 EI_y}{(K_y L_y)^2} = \frac{\pi^2 (210,000) 10122763.6}{(0.9 \times 1810)^2} = 7906348.8 \text{ (N)}$$

$$\bar{\lambda}_y = \sqrt{\frac{A_{eff}f_{yb}}{N_{cr,y}}} = \sqrt{\frac{2186.6 \times 355}{7906348.8}} = 0.313$$

For box section, the buckling curve is c and $\alpha = 0.49$

$$\Phi_y = 0.5[1 + \alpha(\bar{\lambda}_y - 0.2) + \bar{\lambda}_y^2] = 0.5[1 + 0.49(0.313 - 0.2) + 0.313^2] = 0.577$$

$$\chi_y = \frac{1}{\Phi_y + \sqrt{\Phi_y^2 - \bar{\lambda}_y^2}} = \frac{1}{0.577 + \sqrt{0.577^2 - 0.313^2}} = 0.94233$$

$$N_{b,Rd} = \frac{\chi_y A_{eff} f_{yb}}{\gamma_{M1}} = \frac{0.94233 \times 2186.6 \times 355}{1.0} = 731477 \text{ (N)}$$

Member Flexural Capacity: the flexural capacity is calculated considering the limit state of local buckling. Lateral-torsional and distortional buckling are not considered because it is closed-box section.

1. Local buckling:

The effective width method is utilized to calculate the nominal flexural strength in consideration of local and distortional buckling with the compressive stress in the top flange of $f_{yb} = 355 \text{ (N/mm}^2\text{)}$. As the section is subjected to positive moment, the top flange is under compression and it is considered as internal stiffened element:

$$\psi = 1$$

$$k_\sigma = 4$$

$$\varepsilon = \sqrt{\frac{235}{f_{yb} [N/mm^2]}} = \sqrt{\frac{235}{355}} = 0.8136$$

$$\bar{\lambda}_{p,b} = \frac{b_p/t}{28.4\varepsilon\sqrt{k_\sigma}} = \frac{156/4}{28.4 \times 0.8136\sqrt{4}} = 0.844 > 0.673$$

$$\rho = \frac{\bar{\lambda}_{p,b} - 0.055(3 + \psi)}{\bar{\lambda}_{p,b}^2} = \frac{0.844 - 0.055(3 + 1)}{0.844^2} = 0.876 \leq 1.0$$

$$b_{eff} = \rho b_p = 0.876 \times 156 = 136.66 \text{ (mm)}$$

The bottom flange is in tension and calculation of its effective width is not carried out.

The neutral axis of the section with effective flange width and gross web is:

$$\bar{y} = \frac{\sum_i A_i y_i}{A} = \frac{2th\frac{h}{2} + tbh}{t(b_{eff} + b + 2h)} = \frac{2 \times 4 \times 156\frac{156}{2} + 4 \times 156 \times 156}{4(136.66 + 156 + 2 \times 156)} = 80.494 \text{ (mm)}$$

The web is considered as internal (stiffened) element under stress gradient:

$$\sigma_1 = f_{yb} = 355 \text{ (N/mm}^2\text{)}$$

$$\sigma_2 = -f_{yb} \frac{156 - 80.494}{80.494} = -333 \text{ (N/mm}^2\text{)}$$

$$\psi = \frac{\sigma_2}{\sigma_1} = -\frac{333}{355} = -0.938$$

$$k_\sigma = 7.81 - 6.29\psi + 9.78\psi^2 = 7.81 - 6.29 \times (-0.938) + 9.78 \times (-0.938)^2 = 22.315$$

$$\bar{\lambda}_{p,b} = \frac{h_p/t}{28.4\varepsilon\sqrt{k_\sigma}} = \frac{156/4}{28.4 \times 0.8136\sqrt{22.315}} = 0.357 < 0.673 \rightarrow \rho = 1.0$$

$$h_c = \frac{h_p}{(1 - \psi)} = \frac{156}{(1 + 0.938)} = 80.5 \text{ (mm)}$$

$$h_t = h_p - h_c = 156 - 80.5 = 75.5 \text{ (mm)}$$

$$h_{eff} = \rho \frac{h_p}{(1 - \psi)} = 1.0 \times \frac{156}{(1 + 0.938)} = 80.5 \text{ (mm)}$$

$$h_{e1} = 0.4h_{eff} = 0.4 \times 80.5 = 32.2 \text{ (mm)}$$

$$h_{e2} = 0.6h_{eff} = 0.6 \times 80.5 = 48.3 \text{ (mm)}$$

The neutral axis of the section with effective top flange and lip measured from the centerline of the top flange is:

$$\bar{y} = \frac{\sum_i A_i y_i}{A} = 80.5 \text{ (mm)}$$

$$W_{eff,c} = 119723.4 \text{ (mm}^4\text{)}$$

$$W_{eff,t} = 127643.1 \text{ (mm}^4\text{)}$$

$$M_{c,Rd} = \frac{W_{eff,c} f_{yb}}{\gamma_{M0}} = \frac{119723.4 \times 355}{1.0} = 42501807 \text{ (N - mm)}$$

Lateral-torsional buckling is not required for box section, so χ_{LT} is taken as unity. The member buckling moment strength is:

$$M_{b,Rd} = \chi_{LT} W_{eff,c} \frac{f_{yb}}{\gamma_{M1}} = 1.0 \times 119723.4 \frac{355}{1.0} = 42501807 \text{ (N - mm)}$$

Member Shear Capacity:

$$\bar{\lambda}_w = 0.346 \frac{s_w}{t} \sqrt{\frac{f_{yb}}{E}} = 0.346 \frac{156}{4} \sqrt{\frac{355}{210000}} = 0.555 < 0.83$$

$$f_{bv} = 0.58 f_{yb} = 0.58 \times 355 = 205.9 \text{ (N/mm}^2\text{)}$$

$$V_{b,Rd} = \frac{2h_w t f_{bv}}{\gamma_{M0}} = \frac{2 \times 156 \times 4 \times 205.9}{1.0} = 256963 \text{ (N)}$$

Combined D/C ratio:

By observation, Equation 6.36 in Eurocode 3 1-3 2006 would provide the largest D/C ratio and govern the design

Eq. 6.36 in EC3 1-3:

$$\frac{D}{C} = \left(\frac{N_{Ed}}{N_{b,Rd}} \right)^{0.8} + \left(\frac{M_{y,Ed} + \Delta M_{y,Ed}}{M_{by,Rd}} \right)^{0.8} + \left(\frac{M_{z,Ed} + \Delta M_{z,Ed}}{M_{bz,Rd}} \right)^{0.8}$$

$$= \left(\frac{400,000}{731477} \right)^{0.8} + \left(\frac{8190250 + 0}{42501807} \right)^{0.8} + 0.0^{0.8} = 0.885$$